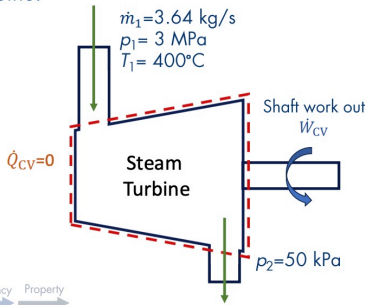


## Example (Part 1 of 2)

Steam enters a steady, adiabatic turbine at 3 MPa and 400°C and leaves at 50 kPa. If the mass flow rate of the water is 3.64 kg/s, what is maximum possible power output of an idealized, reversible turbine.



10

$$\frac{dS_{cv}}{dt} = \sum \frac{\dot{Q}_k}{T_k} + \dot{m}(s_{in} - s_{out}) + \dot{S}_{gen}$$

steady-state      adiabatic      reversible

"Isentropic" (adiabatic, reversible)

State 1:  $P_1 = 3 \text{ MPa}$  → Table A-6 →  $s_1 = 6.9235 \text{ [kJ/kg} \cdot \text{K]}$   
 $T_1 = 400^\circ \text{C}$  → Superheated vapor →  $h_1 = 3231.7 \text{ [kJ/kg]}$

State 2:  $P_2 = 50 \text{ kPa}$  → Table A-5 →  $x_2 = \frac{s_2 - s_f}{s_g} = \frac{6.9235 - 1.0912 \text{ [kJ/kg} \cdot \text{K}]}{6.5019 \text{ [kJ/kg} \cdot \text{K]}}$   
 $s_2 = s_1$  → 2-phase mixture →  $= 0.817$   
"ideal" = isentropic

$$h_2 = h_f + x h_g$$

$$= 340.54 \text{ [kJ/kg]} + 0.817 \times 2304.7 \text{ [kJ/kg]}$$

$$= 2407.9 \text{ [kJ/kg]}$$

First law:  $\frac{dE}{dt} = \dot{Q}_{cv} - \dot{W}_{cv} + \dot{m}(h + \frac{1}{2}v^2 + gz)_1 - \dot{m}(h + \frac{1}{2}v^2 + gz)_2$

steady      adiabatic      negligible

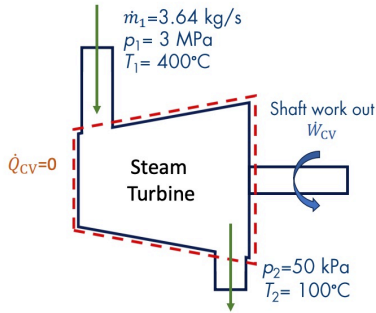
$$\dot{W}_{cv} = \dot{m}(h_1 - h_2)$$

$$= 3.64 \text{ [kg/s]} \times (3231.7 \text{ [kJ/kg]} - 2407.9 \text{ [kJ/kg]}) = \boxed{3.00 \text{ MW}}$$

Ideal, reversible power output of turbine

## Example (Part 2 of 2)

The actual steam turbine from Part 1 is not ideal. The water exits the turbine at  $100^\circ\text{C}$ . What is the isentropic efficiency of the real turbine.



Re-cap → Open → Efficiency → Property

12

The previous State 2 was idealized.

$$\text{State } 2a \text{ (actual)} : \begin{array}{l} p_{2a} = 50 \text{ kPa} \\ T_{2a} = 100^\circ\text{C} \end{array} \rightarrow \text{Table A-6} \rightarrow h_{2a} = 2682.4 \text{ [kJ/kg]}$$

$$\eta = \frac{w_a}{w_s} = \frac{h_1 - h_{2a}}{h_1 - h_{2s}} = \frac{3237.7 - 2682.4 \text{ [kJ/kg]}}{3237.7 - 2402.9 \text{ [kJ/kg]}} = 66.7\%$$